

Control Theory and Congestion

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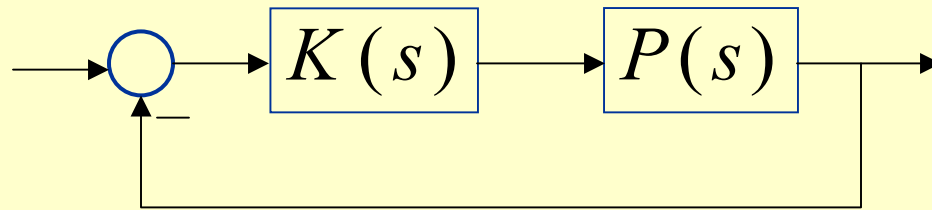
Cambridge/Caltech and UCLA

IPAM Tutorial – March 2002.

Outline of second part:

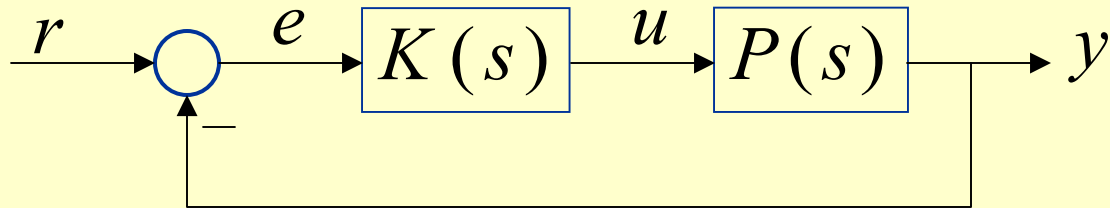
1. Performance in feedback loops: tracking, disturbance rejection, transient response. Integral control.
2. Fundamental design tradeoffs. The role of delay. Bode Integral formula
3. Extensions to multivariable control.

Performance of feedback loops



- Stability and its robustness are essential properties; however, they are only half of the story.
- The closed loop system must also satisfy some notion of performance:
 - Steady-state considerations (e.g. tracking errors).
 - Disturbance rejection.
 - Speed of response (transients, bandwidth of tracking).
- Performance and stability/robustness are often at odds.
- For single input-output systems, frequency domain tools (Nyquist, Bode) are well suited for handling this tradeoff.

Performance specs 1: Steady-state tracking



$$e(t) = r(t) - y(t)$$

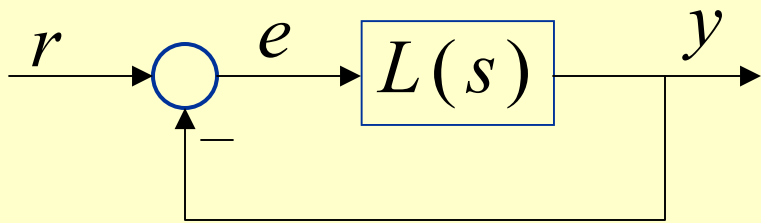
Error between reference signal r and output y .

Tracking means this error is kept small.

Suppose that $r(t) = r_0$, constant, and that the system is stable. Then as $t \rightarrow \infty$, $e(t) \rightarrow e(\infty)$, **steady-state error**.

Ideally, we would like the steady-state error to be zero.

Tracking, sensitivity and loop gain



The mapping from $r(t)$ to $e(t)$ has transfer function $S(s) = \frac{1}{1 + L(s)}$.

That is, $R(s) = S(s)E(s)$ in Laplace.

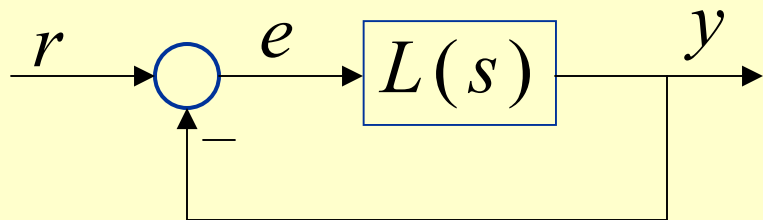
$S(s)$ is called the **sensitivity function** of the system.

Under stability, $S(s)$ has no poles in $\text{Re}[s] \geq 0$.

Then for $r(t) \equiv r_0$, we have $e(\infty) = S(0)r_0 = \frac{1}{1 + L(0)}r_0$

Good steady-state tracking $\leftrightarrow S(0)$ small $\leftrightarrow L(0)$ large.

Integral control



Suppose $L(s)$ has a pole at $s = 0$.

$$\text{Then } S(0) = \frac{1}{1 + L(0)} = 0.$$

Zero steady-state error!

$$\text{Example: } L(s) = \frac{K}{s}. \quad \dot{y}(t) = K(r - y).$$

Loop is stable for $K > 0$, and has a pole at $s = 0$.

Therefore, it has zero steady-state tracking error.

$$\text{In the time domain: for } r(t) \equiv r_0, \quad \boxed{y(t) = r_0 \left(1 - e^{-Kt}\right) \rightarrow r_0}$$

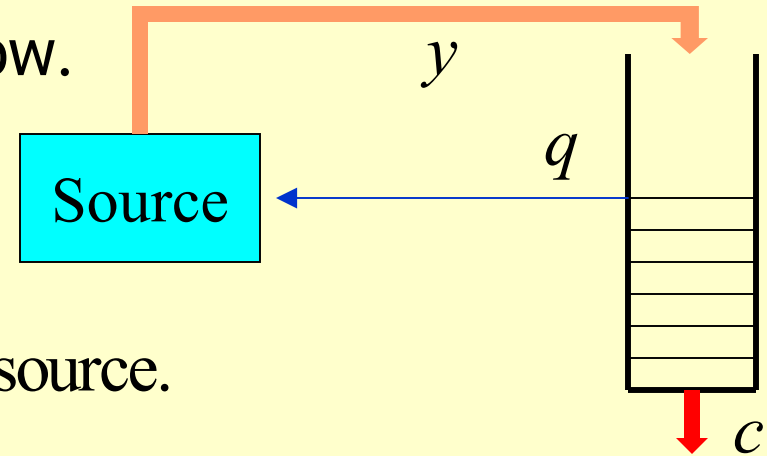
Simple congestion control example

Single link/source, no delays for now.

y : Transmission rate (pkts/sec)

c : Capacity of the link (pkts/sec)

q : Queue size; assume it is fed back to source.



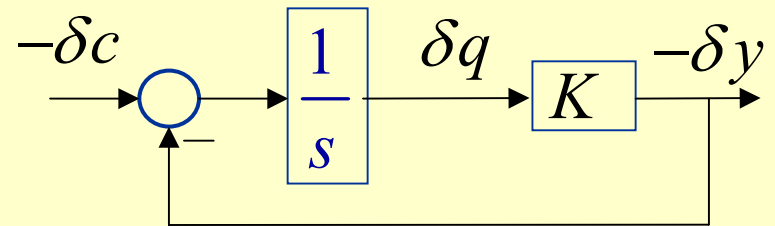
Suppose source control is $y = f(q)$, where f is a decreasing function.

Model: $\dot{q} = y - c = f(q) - c$ Equilibrium for $c = c_0$: $f(q_0) = y_0 = c_0$.

Linearize around it: $c = c_0 + \delta c$, $y = y_0 + \delta y$, $q = q_0 + \delta q$.

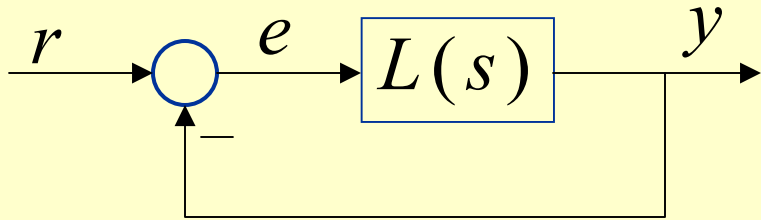
$$\delta y = -K \delta q, \quad K = -\frac{\partial f}{\partial q}(q_0) > 0.$$

$$\delta \dot{q} = \delta y - \delta c.$$



Integral Control \Rightarrow Perfect steady-state tracking for constant δc .

Performance specs 2: tracking of low-frequency reference signals.



Transfer function from $r(t)$ to $e(t)$ is the sensitivity $S(s) = \frac{1}{1 + L(s)}$.

Let $S(j\omega) = |S(j\omega)| e^{j\phi_s(\omega)}$ be the polar decomposition.

Assume the system is stable: then the steady-state response to a sinusoidal reference $r(t) = r_0 \cos(\omega_0 t)$ is $e(t) = r_0 |S(j\omega_0)| \cos(\omega_0 t + \phi_s(\omega_0))$.

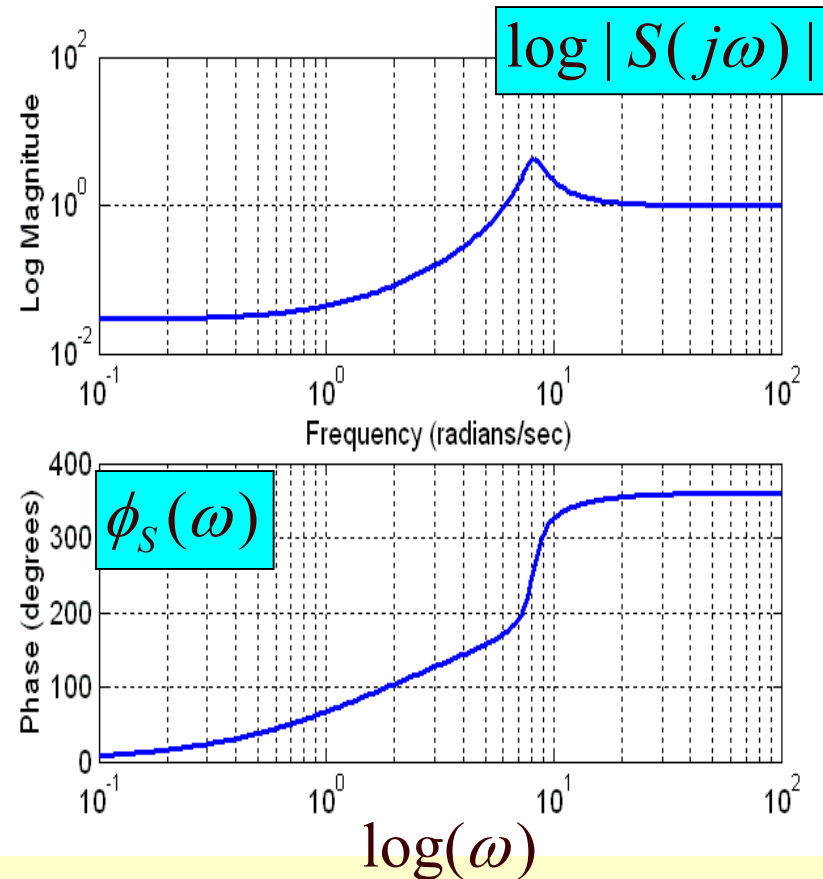
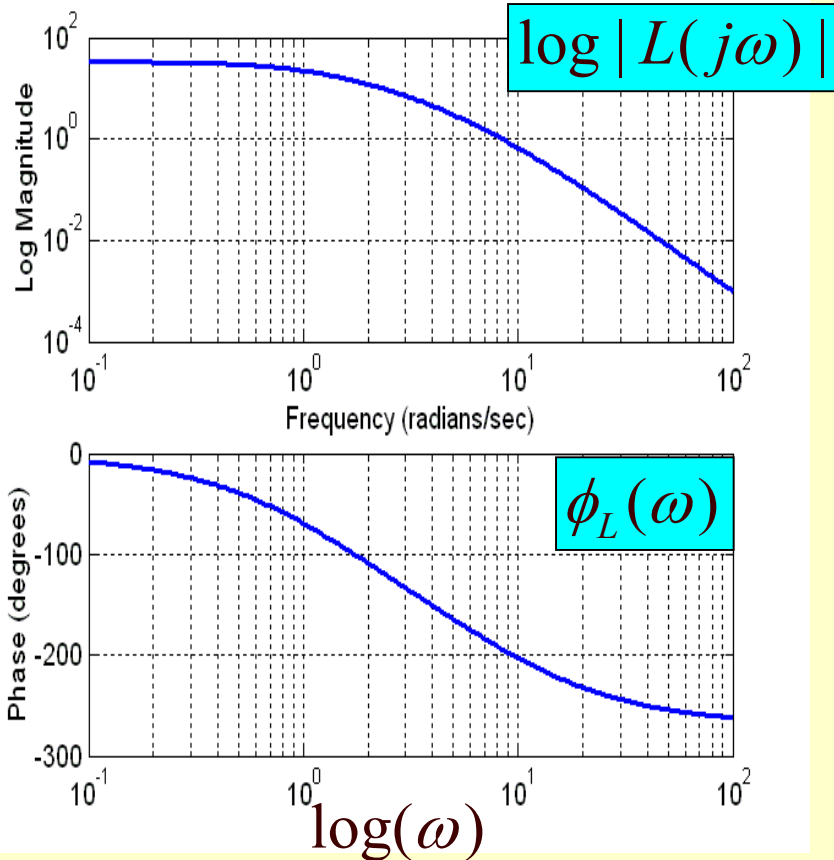
Good steady-state tracking $\leftrightarrow |S(j\omega_0)|$ small $\xrightarrow{\approx} |L(j\omega_0)|$ large.

Representation of frequency functions

$L(j\omega)$

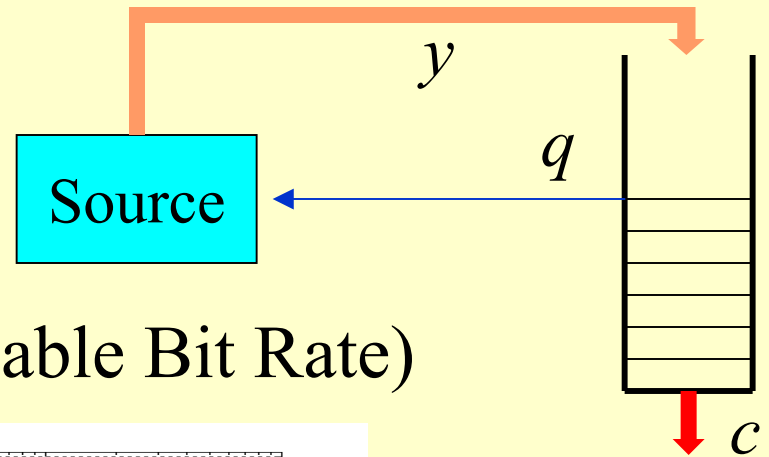
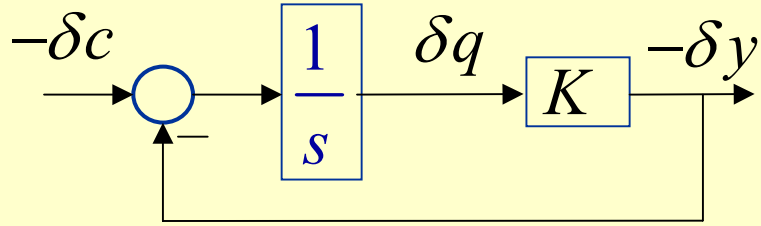
Bode plot

$S(j\omega)$



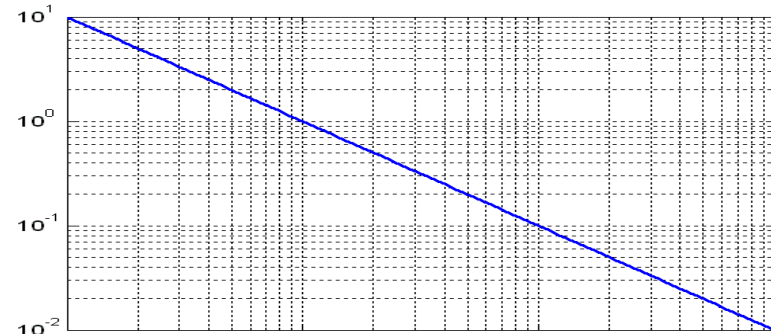
Tracking \leftrightarrow Large $|L(j\omega)| \leftrightarrow$ Small $|S(j\omega)|$
in frequency range of interest.

Example 2: tracking of variable references



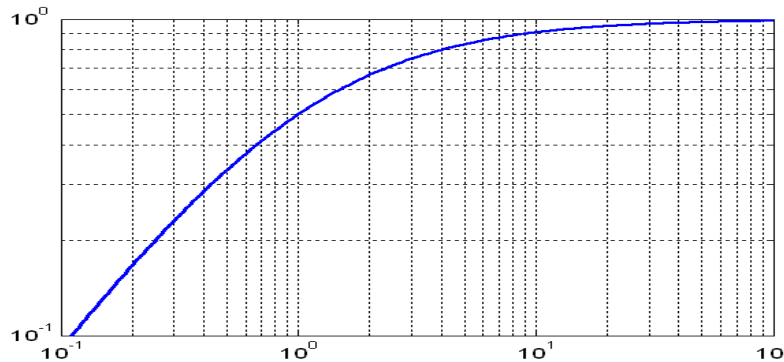
Variations in capacity (e.g. Available Bit Rate)

$$\log |L(j\omega)|$$



$$L(j\omega) = \frac{K}{j\omega}$$

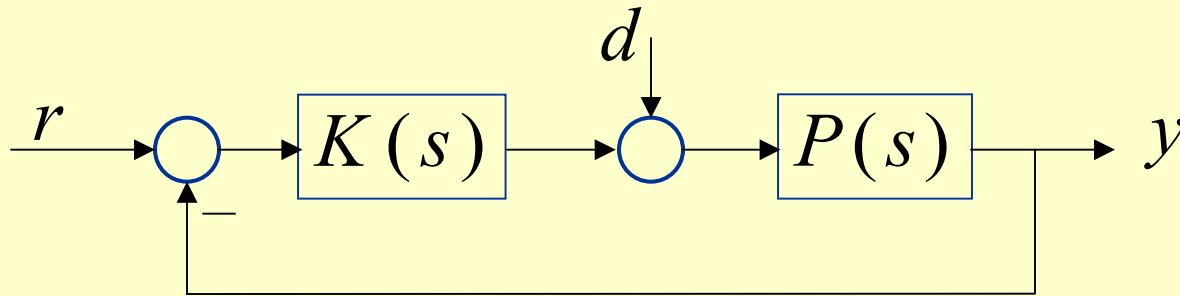
$$\log |S(j\omega)|$$



As K grows,
track higher
bandwidth

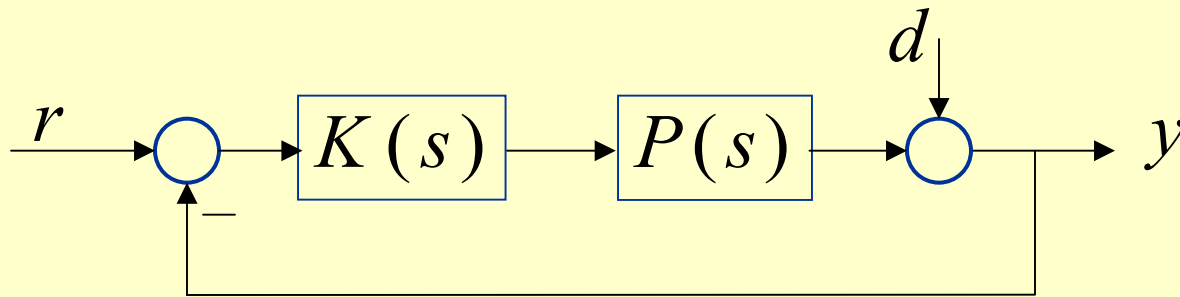
$\log \omega$

Performance specs 3: disturbance rejection.



Input disturbance:

$$T_{d \rightarrow y}(s) = \frac{P(s)}{1 + L(s)}$$

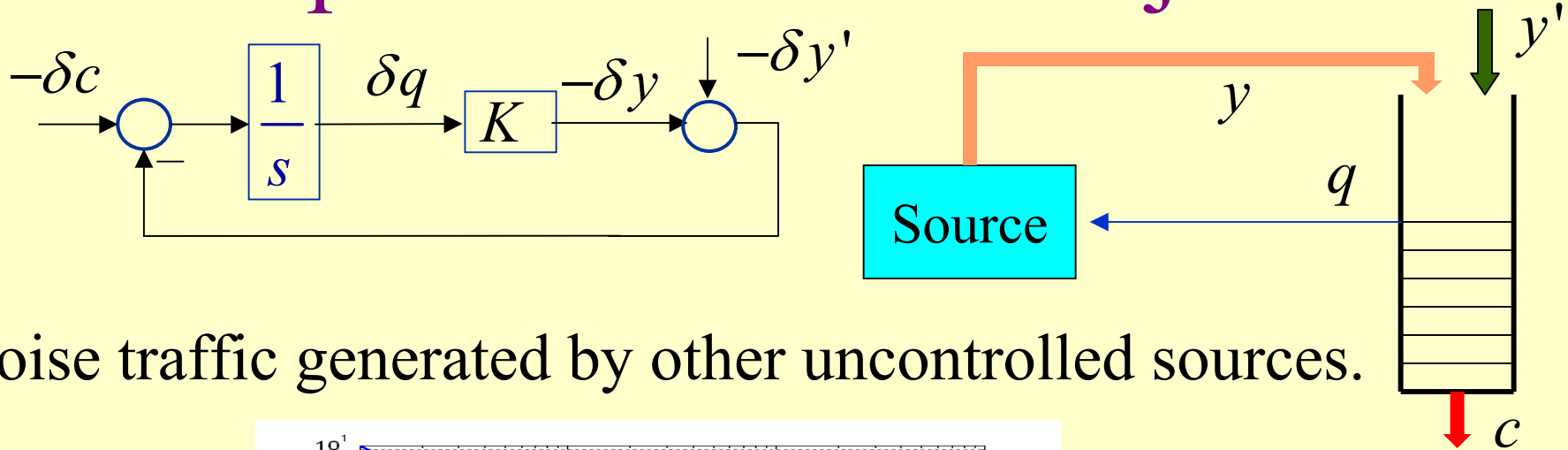


Output disturbance:

$$T_{d \rightarrow y}(s) = \frac{1}{1 + L(s)}$$

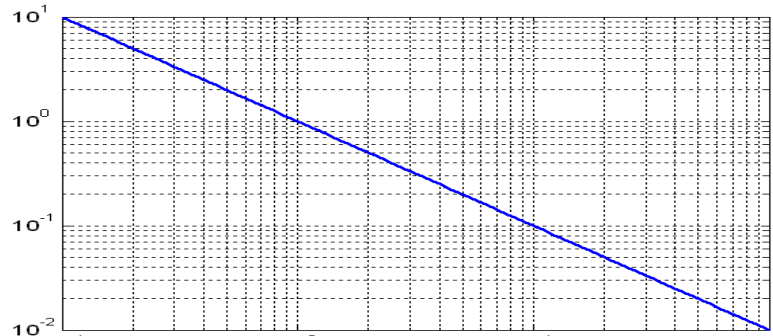
To reject disturbances, we need attenuation in the frequency range of interest \approx Large $|L(j\omega)|$.

Example 3: disturbance rejection



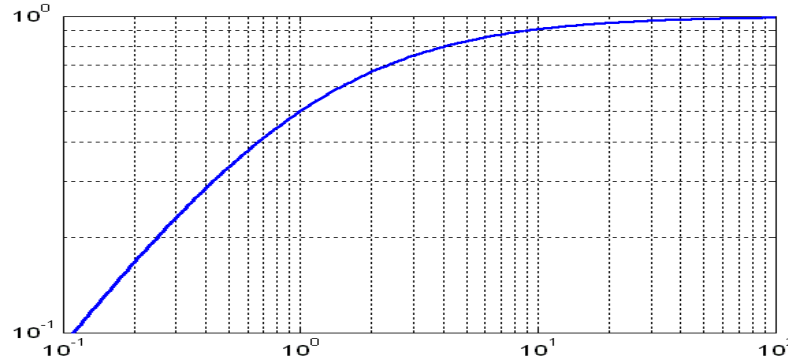
Noise traffic generated by other uncontrolled sources.

$\log |L(j\omega)|$



$$L(j\omega) = \frac{K}{j\omega}$$

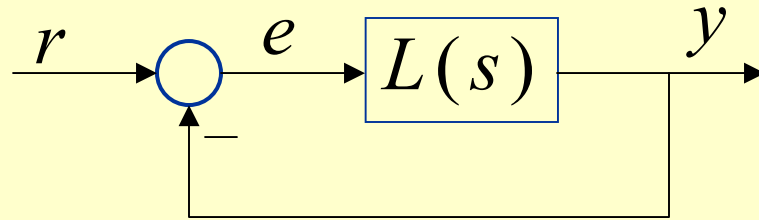
$\log |S(j\omega)|$



As K grows, reject disturbances over a higher bandwidth

$\log \omega$

Performance specs 4: speed of response

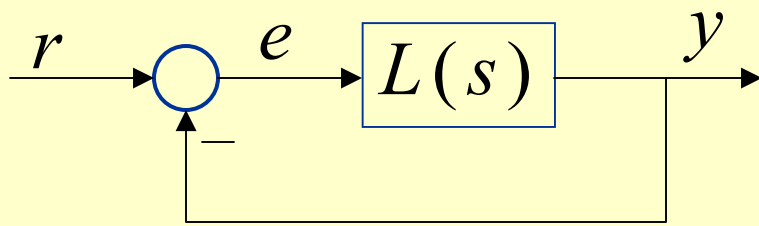


- Superimposed to the steady-state solutions discussed before, we have transient terms of the form $\sum_i C_i e^{-s_i t}$. Here the modes s_i are the roots of $1 + L(s) = 0$.
- For fast response, $\text{Re}[s_i]$ must be as negative as possible.

Example: $L(s) = \frac{K}{s}$. $1 + L(s) = 0 \leftrightarrow s_1 = -K$.

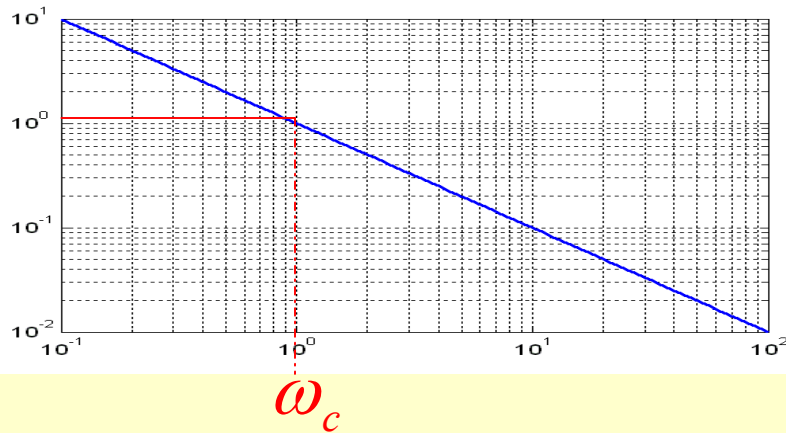
The higher K , the faster our transient response.

For instance for $r(t) \equiv r_0$, solution is $y(t) = r_0 \left(1 - e^{-Kt}\right)$



Heuristic look based on Fourier:
 frequencies where $|L(j\omega)| \ll 1$
 cannot occur (filtered out). So
 the speed of response is roughly
 the bandwidth where $|L(j\omega)| \geq 1$.

$\log |L(j\omega)|$



Transient decays in a
 time of the order of $\frac{1}{\omega_c}$

For $L(s) = \frac{K}{s}$ (e.g. our congestion control with queue feedback)

$\omega_c = K \rightarrow$ decays in the order of $\frac{1}{K}$ seconds.

For faster response, increase the open loop bandwidth.

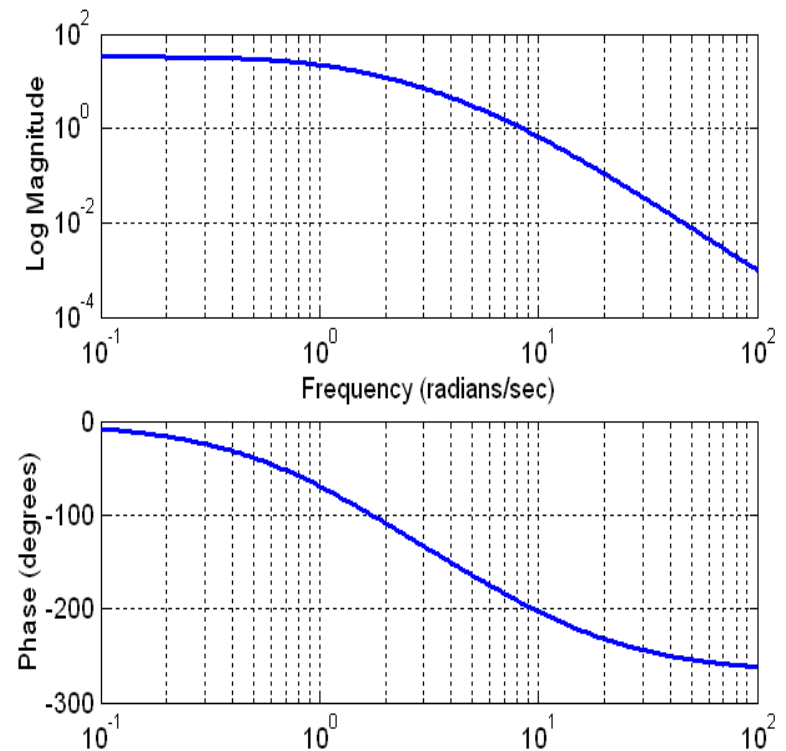
Performance specifications: recap

- Tracking of constant, or varying reference signals.
- Disturbance rejection.
- Transient response.

Rule of thumb for all: increase the gain or bandwidth of the loop transfer function $L(j\omega)$.

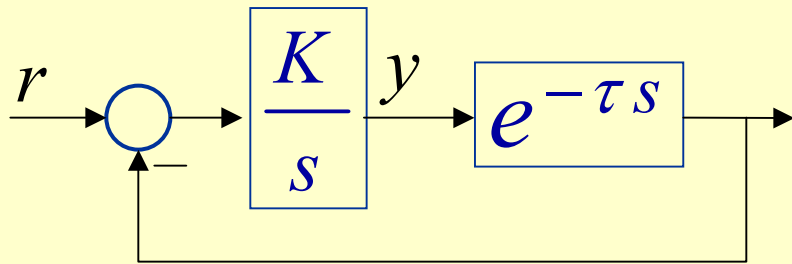
What stops us from arbitrarily good performance?

Answer: stability/robustness.



Example: loop with integrator and delay.

$$\dot{y}(t) = K(r(t) - y(t - \tau))$$



$$L(s) = \frac{Ke^{-\tau s}}{s}$$

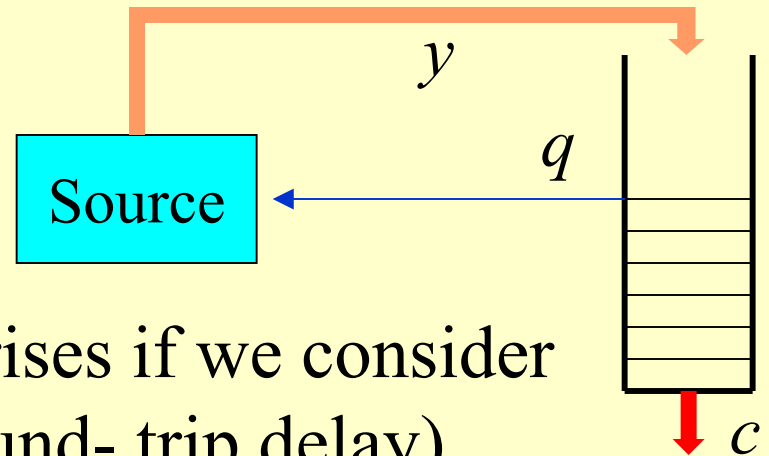
$$L(j\omega) = \frac{Ke^{-\tau j\omega}}{j\omega} \Rightarrow |L(j\omega)| = \frac{K}{|\omega|} \quad (\text{independent of delay!})$$

Our earlier rule says: increase K for performance.

Stability? $1 + L(s) = 0 \leftrightarrow s + Ke^{-\tau s} = 0$.

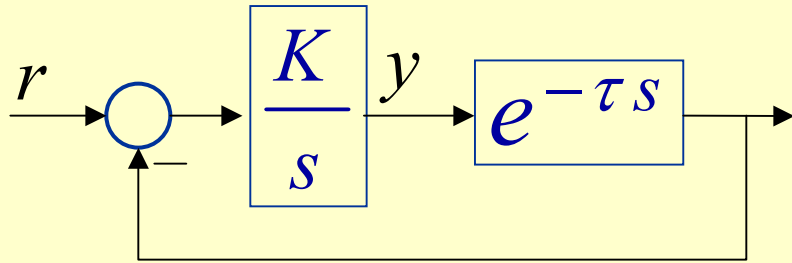
Transcendental equation. However, use Nyquist.

Example:



(arises if we consider round-trip delay)

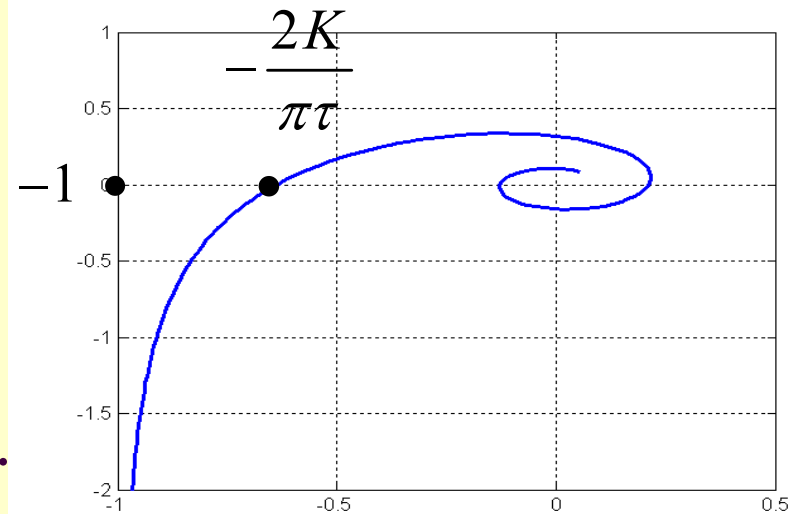
Stability analysis via Nyquist:



$$\text{Loop function } L(s) = \frac{Ke^{-\tau s}}{s}$$

$$|L(j\omega)| = \frac{K}{|\omega|}, \quad \phi(\omega) = -\frac{\pi}{2} - \omega\tau.$$

Nyquist plot of $L(j\omega)$:



To avoid encirclements, impose

$$|L(j\omega_0)| < 1 \text{ at } \omega_0 \text{ where } \phi(\omega_0) = -\pi$$

Stable for

$$K < \frac{\pi}{2\tau}$$

Not much harder than analysis without delay!

Much simpler than other alternatives

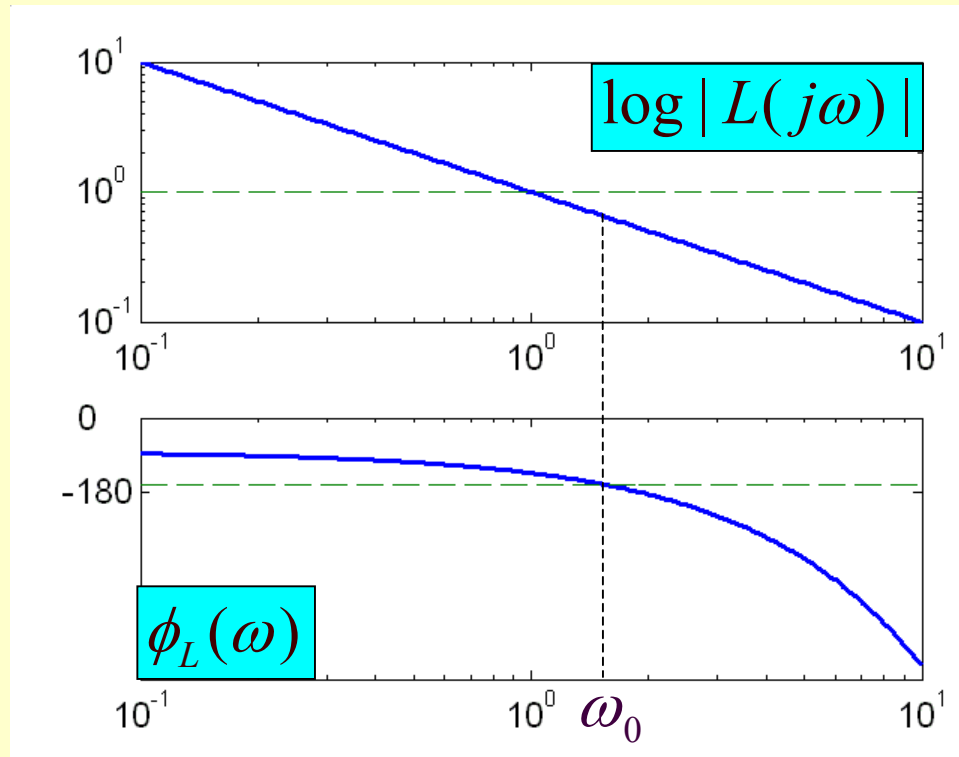
(transcendental equations, Lyapunov functionals,...)

Stability in the Bode plot

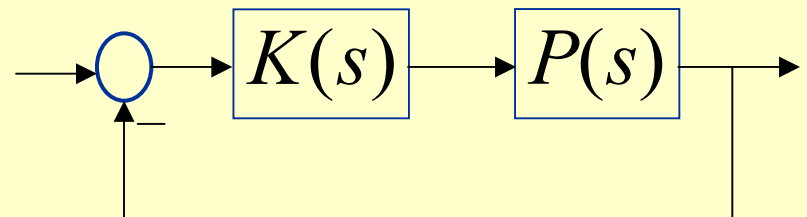
$$|L(j\omega)| = \frac{K}{|\omega|}, \quad \phi(\omega) = -\frac{\pi}{2} - \omega\tau.$$

Impose $|L(j\omega_0)| < 1$
at ω_0 : $\phi(\omega_0) = -\pi$

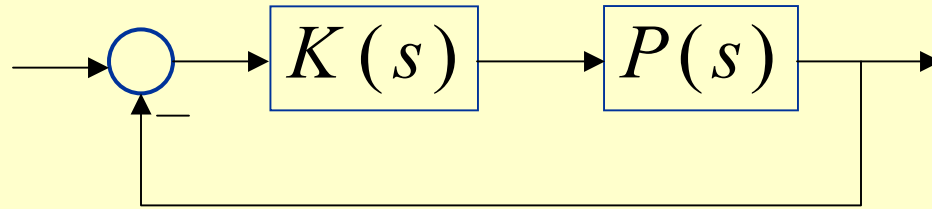
Increasing K moves
the top plot upwards.
 \Rightarrow Constraint on K
for stability.



Conclusion: delay limits the achievable performance.
Also, other dynamics of the plant (known or uncertain)
produce a similar effect.

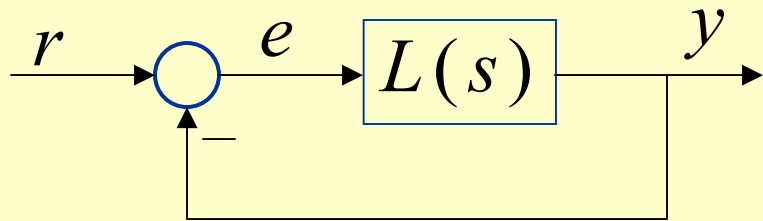


The performance/robustness tradeoff



- As we have seen, we can improve performance by increasing the gain and bandwidth of the loop transfer function $L(j\omega)$.
- $L(s)$ can be designed through $K(s)$. By canceling off $P(s)$, one could think $L(s)$ would be arbitrarily chosen. However:
 - Unstable dynamics cannot be canceled.
 - Delay cannot be canceled (otherwise $K(s)$ would not be causal).
 - Cancellation is not robust to variations in $P(s)$.
- Therefore, the given plant poses essential limits to the performance that can be achieved through feedback.
- Good designs address this basic tradeoff. For single I-O systems, “loopshaping” the Bode plot is an effective method.

The Bode Integral formula



Recall: the mapping from $r(t)$ to $e(t)$

has transfer function $S(s) = \frac{1}{1 + L(s)}$.

For tracking, we want the sensitivity $|S(j\omega)|$ to be small, for as large a frequency range as possible. How large can it be?

Theorem (Bode): Suppose $L(s) = \frac{n(s)}{d(s)}$, a rational function

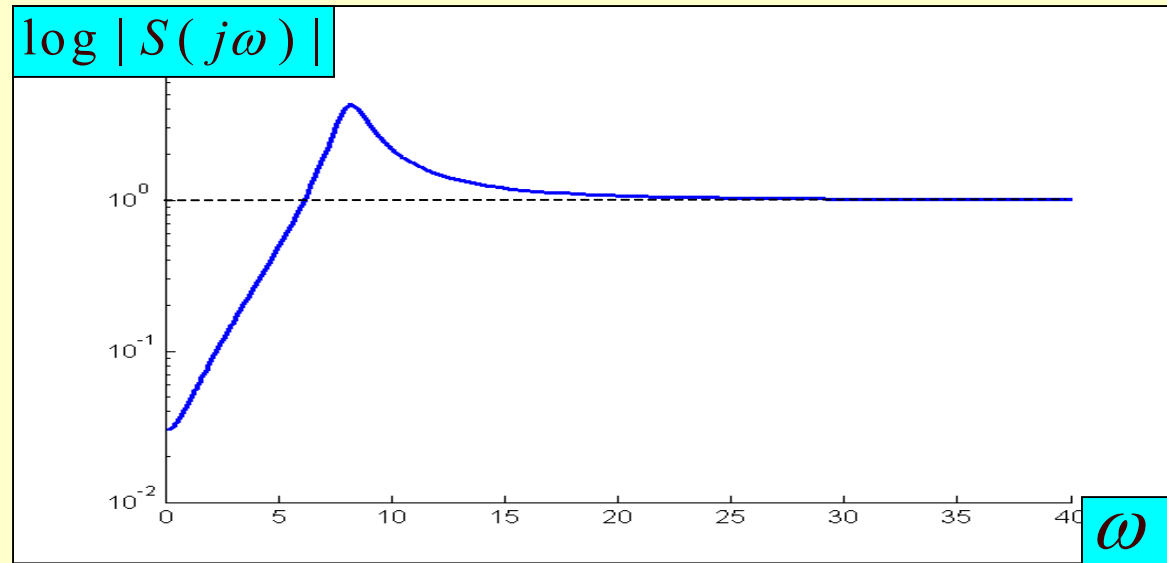
with $\deg(d(s)) - \deg(n(s)) \geq 2$.

Let $\{p_i\}$ be the set of poles of $L(s)$ in $\text{Re}[s] > 0$. Then

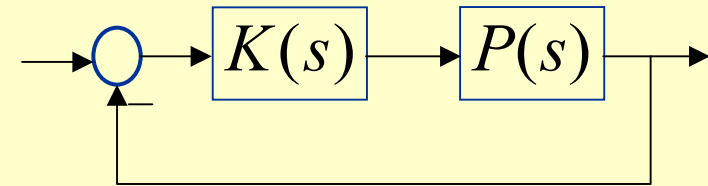
$$\int_0^{\infty} \log |S(j\omega)| d\omega = \pi \log(e) \sum \text{Re}[p_i]$$

The Bode Integral formula.

$$\int_0^{\infty} \log |S(j\omega)| d\omega = \pi \log(e) \sum \operatorname{Re}[p_i]$$



The unstable poles p_i that come from the plant $P(s)$ cannot be eliminated by $K(s)$
 \Rightarrow Integral of sensitivity is a conserved quantity over all stabilizing feedbacks.

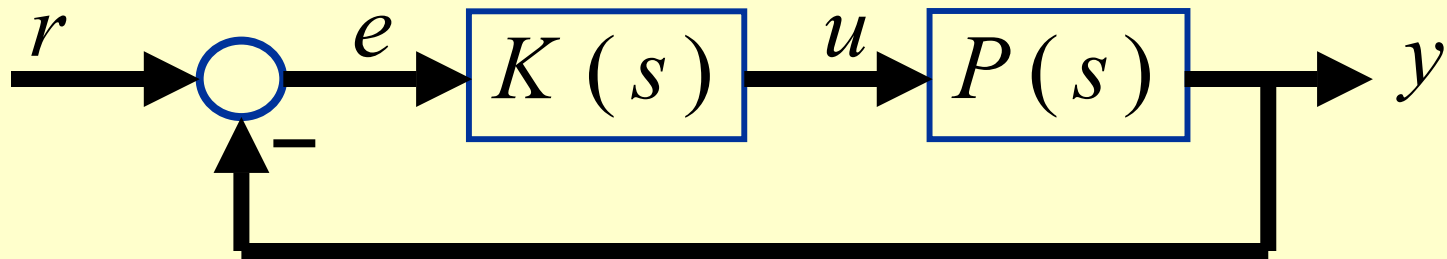


Small sensitivity at low frequencies must be "paid" by a larger than 1 sensitivity at some higher frequencies.

But all this is only linear!

- The above tradeoff is of course present in nonlinear systems, but harder to characterize, due to the lack of a frequency domain (partial extensions exist).
- So most successful designs are linear based, followed up by nonlinear analysis or simulation.
- Beware of claims of superiority of “truly nonlinear” designs. They rarely address this tradeoff, so may have poor performance or poor robustness (or both).
- A basic test: linearized around equilibrium, the nonlinear controller should not be worse than a linear design.

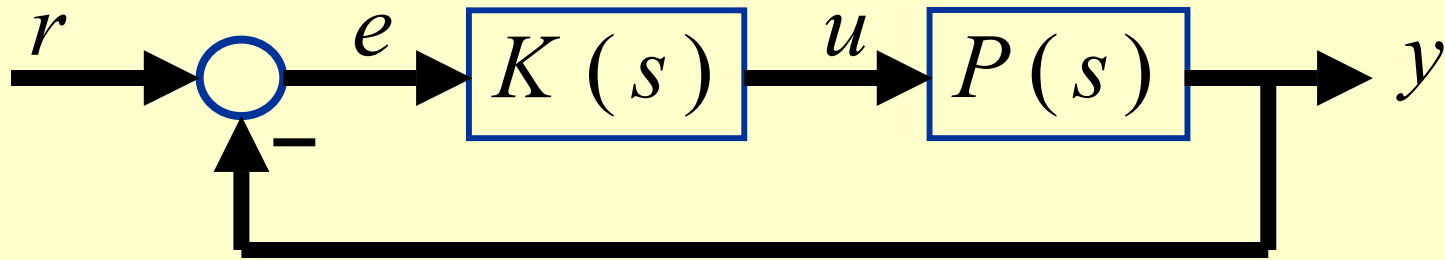
Multivariable control



Signals are now vector-valued (many inputs and outputs).
Transfer functions are matrix-valued.

$$y(s) = \begin{bmatrix} y_1(s) \\ \vdots \\ y_m(s) \end{bmatrix} = \begin{bmatrix} P_{11}(s) & \cdots & P_{1n}(s) \\ \vdots & \ddots & \vdots \\ P_{m1}(s) & \cdots & P_{mn}(s) \end{bmatrix} \begin{bmatrix} u_1(s) \\ \vdots \\ u_n(s) \end{bmatrix} = P(s)u(s)$$

$$\left. \begin{array}{l} y(s) = \overbrace{P(s)K(s)}^{L(s)} e(s) \\ e(s) = r(s) - y(s) \end{array} \right\} \Rightarrow [I + L(s)]e(s) = r(s)$$



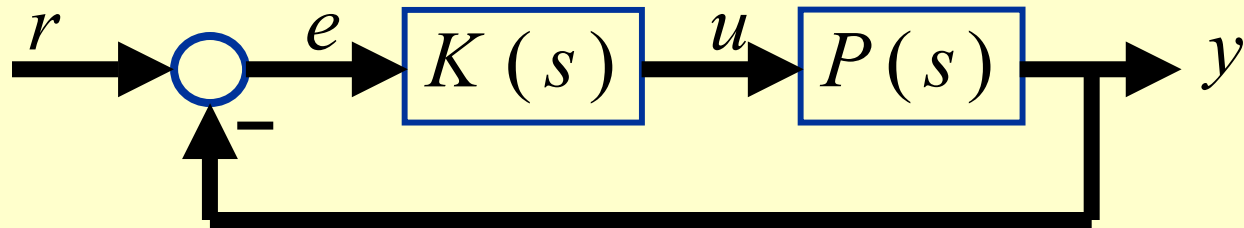
$$e(s) = \underbrace{[I + L(s)]^{-1}}_{S(s)} r(s)$$

$$y(s) = L(s)[I + L(s)]^{-1} r(s)$$

Stability: poles of $[I + L(s)]^{-1}$ (i.e., roots of $\det[I + L(s)] = 0$) must have negative real part.

Multivariable Nyquist criterion: study encirclements of the origin of $\det[I + L(j\omega)] = \prod (1 + \lambda_i(j\omega))$, where $\lambda_i(j\omega)$ are the eigenvalues of $L(j\omega)$.

Performance of multivariable loops



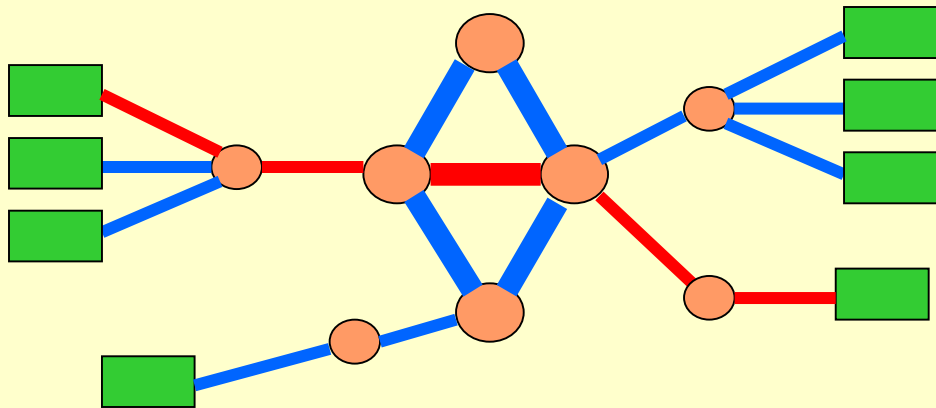
$$e(j\omega) = S(j\omega) r(j\omega) = [I + L(j\omega)]^{-1} r(j\omega)$$

The tracking error will depend on frequency, but also on the direction of the vector $r(j\omega)$. The worst-case direction is captured by the maximum singular value:

$$\overline{\sigma}(S(j\omega)) = \max \{ |S(j\omega)v| : v \in \mathbb{C}^n, |v| = 1 \}.$$

Network congestion control example

L communication links shared by S source-destination pairs.



Routing matrix:

$$R_{li} = \begin{cases} 1 & \text{if link } l \text{ serves source } i \\ 0 & \text{otherwise} \end{cases}$$

x_i : Rate of i -th source (pkts/sec)

$y_l = \sum_{i \text{ uses } l} x_i$: Total rate of l -th link (pkts/sec)

c_l : Capacity of the l -th link (pkts/sec)

b_l : Backlog of the l -th link (pkts)

$q_i = \sum_{i \text{ uses } l} b_l$: Total backlog for i -th source (pkts)

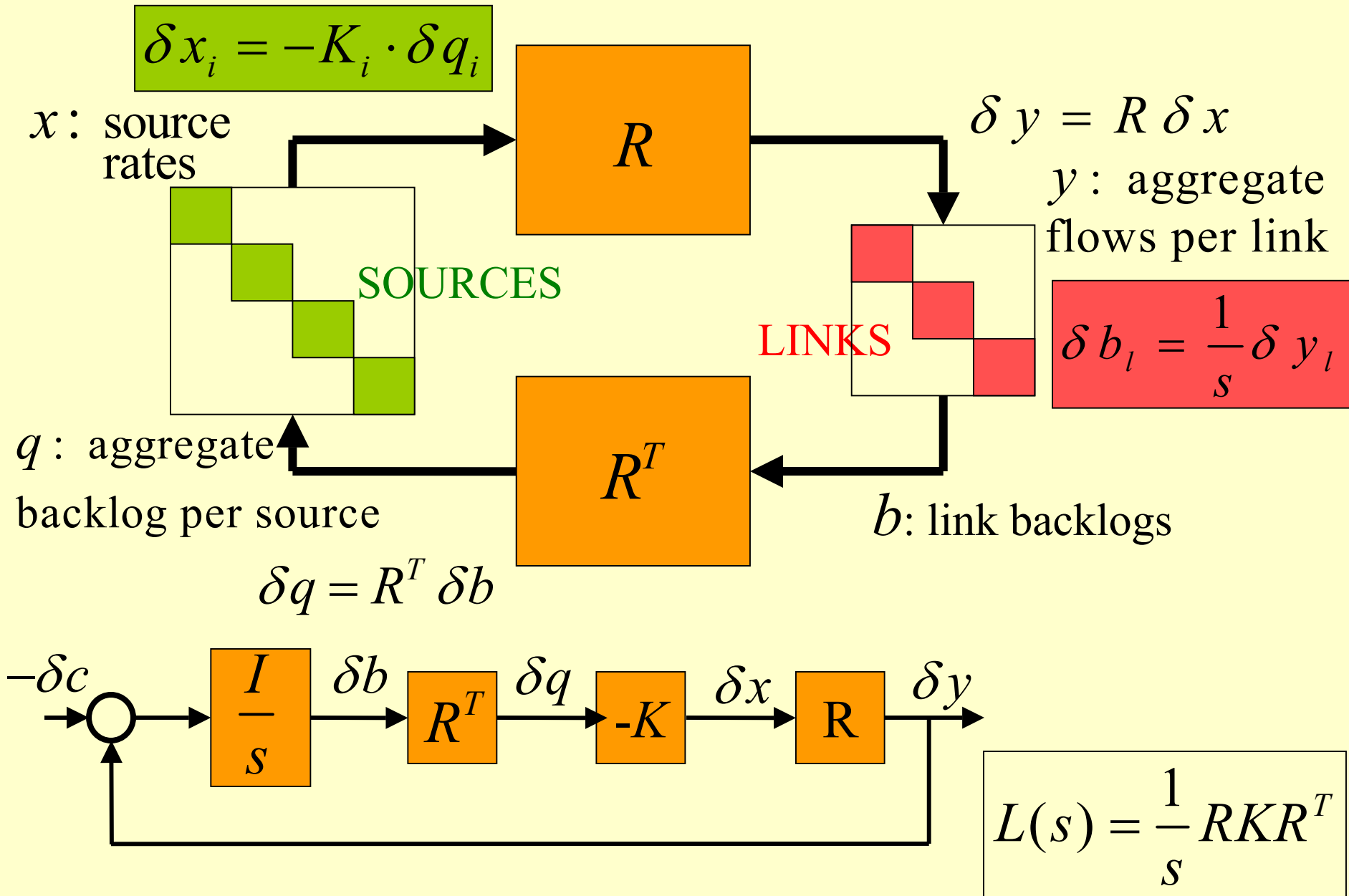
$$y = Rx$$

$$\frac{db_l}{dt} = (y_l - c_l)$$

$$q = R^T b$$

Suppose sources receive q_i by feedback, and set $x_i = f_i(q_i)$

Linearized multivariable model, around equilibrium.



Now: $L(s) = \frac{1}{s} RKR^T$ is easily diagonalized.

$$RKR^T = V^T \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & \lambda_L \end{bmatrix} V, \quad \lambda_l \geq 0 \Rightarrow L(s) = V^T \begin{bmatrix} \frac{\lambda_1}{s} & 0 & 0 \\ 0 & \ddots & 0 \\ 0 & 0 & \frac{\lambda_L}{s} \end{bmatrix} V.$$

- Modes: roots of $\det(I + L(s)) = 0 \rightarrow s = -\lambda_l, \quad l = 1, \dots, L.$

Therefore: stable if RKR^T is full rank. Transient response dominated by slowest mode, $\lambda_{\min}(RKR^T)$.

- Singular values of $S(j\omega) = (I + L(j\omega))^{-1}$ are

$$\left| 1 + \frac{\lambda_l}{j\omega} \right|^{-1} = \left| \frac{\omega}{\lambda_l + j\omega} \right| \Rightarrow \bar{\sigma}(S(j\omega)) = \left| \frac{\omega}{\lambda_{\min} + j\omega} \right|$$

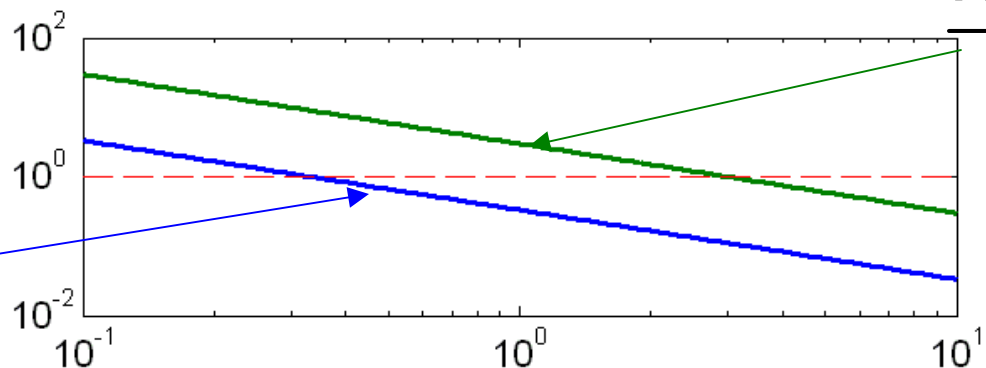
Performance analysis reduces to the scalar case.

Now, consider stability in the presence of delay. For simplicity, use a common delay (RTT) for all loops.

$$L(s) = \frac{e^{-\tau s}}{s} RKR^T$$

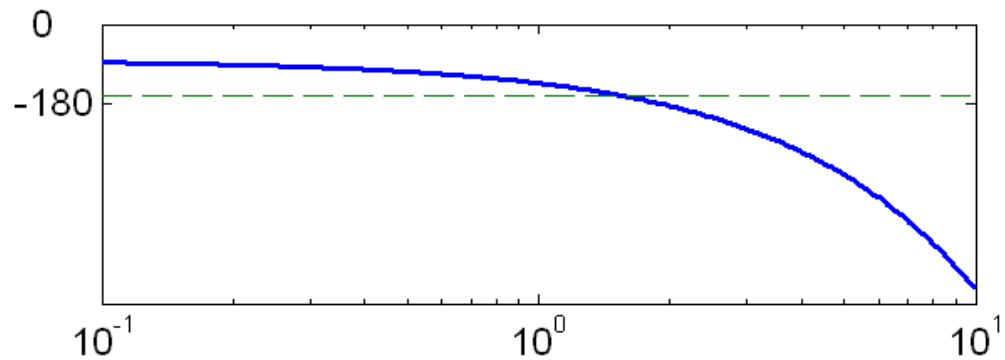
$$\frac{\lambda_{\min}}{\omega}$$

$$\frac{\lambda_{\max}}{\omega}$$



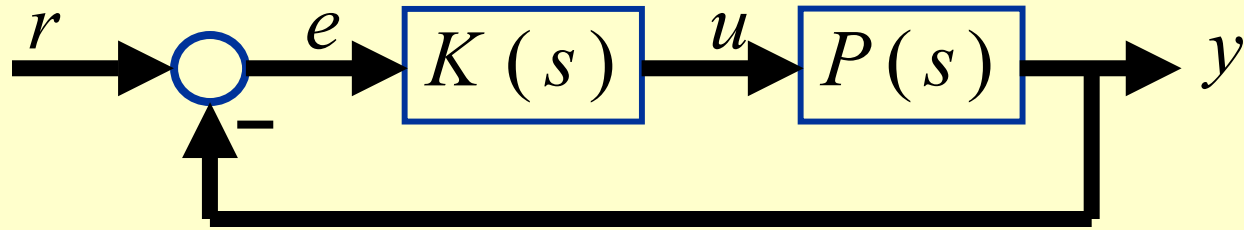
Diagonalize and apply Nyquist: Stable for

$$\lambda_{\max}(RKR^T) < \frac{\pi}{2\tau}$$



Summary: performance defined by $\lambda_{\min}(RKR^T)$, delay robustness by $\lambda_{\max}(RKR^T)$. Tradeoff is harder for ill-conditioned RKR^T !

More generally, eigenvalues don't tell the full story.



- Performance: for transfer functions which are not self-adjoint, $\overline{\sigma}(S(j\omega))$ can be much larger-than the maximum eigenvalue.
- Robust stability: consider a ball of plants $P(s) = P_0(s) + \Delta(s)$, $\overline{\sigma}(\Delta(j\omega)) \leq 1/\alpha(\omega)$. Nyquist not very useful to establish stability for all Δ , since $\det(I + KP)$ depends on it in a complicated way.

However, it can be shown that the condition

$\overline{\sigma}(S(j\omega)K(j\omega)\alpha(\omega)) < 1 \quad \forall \omega$ gives robust stability.

Singular values are more important than eigenvalues.

Summary

- A well designed feedback will respond as quickly as possible to regulate, track references or reject disturbances.
- The fundamental limit to the above features is the potential for instability, and its sensitivity to errors in the model. A good design must balance this tradeoff (robust performance).
- In SISO, linear case, tradeoff is well understood by frequency domain methods. This explains their prevalence in design.
- Nonlinear aspects usually handled a posteriori. Nonlinear control can potentially (but not necessarily) do better. A basic test: linearization around any operating point should match up with linear designs.
- In multivariable systems, frequency domain tools extend with some complications (ill conditioning, singular values versus eigenvalues,...)
- All of this is relevant to network flow control: performance vs delay/robustness, ill-conditioning,... Nonlinearity seems mild.